# Ocean Modeling - EAS 8803 Boundary Conditions

- Type of boundary conditions (e.g. kinematic and dynamic)
- Sumerical implementation
- Surface & bottom boundary condition for momentum
- The Ekman boundary layers

Introduction to Geophysical Fluid Dynamics Physical and Numerical Aspects



Benoit Cushman-Roisin and Jean-Marie Beckers

Academic Press Chapter 4 and 8

### **Boundary Conditions**



**Figure 4-1** Schematic representation of possible exchanges between the system under investigation and the surrounding environment. Boundary conditions must specify the influence of this outside world on the evolution within the domain. Exchanges may take place at the air-sea interface, in bottom layers, along coasts and/or at any other boundary of the domain.

### **Boundary Conditions**

#### state variables (variables

with time derivative)

$$\begin{aligned} x - \text{momentum:} \quad & \left( \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} - fv = \\ & -\frac{1}{\rho_0} \frac{\partial p}{\partial x} + \frac{\partial}{\partial x} \left( \mathcal{A} \frac{\partial u}{\partial x} \right) + \frac{\partial}{\partial y} \left( \mathcal{A} \frac{\partial u}{\partial y} \right) + \frac{\partial}{\partial z} \left( \nu_E \frac{\partial u}{\partial z} \right) \\ y - \text{momentum:} \quad & \left( \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} + fu = \\ & -\frac{1}{\rho_0} \frac{\partial p}{\partial y} + \frac{\partial}{\partial x} \left( \mathcal{A} \frac{\partial v}{\partial x} \right) + \frac{\partial}{\partial y} \left( \mathcal{A} \frac{\partial v}{\partial y} \right) + \frac{\partial}{\partial z} \left( \nu_E \frac{\partial v}{\partial z} \right) \\ z - \text{momentum:} \quad & 0 = - \frac{\rho_P}{\partial z} - \rho_g \\ \text{continuity:} \quad & \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \\ \text{energy:} \quad & \frac{\partial \rho}{\partial t} + u \frac{\partial \rho}{\partial x} + v \frac{\partial \rho}{\partial y} + w \frac{\partial \rho}{\partial z} = \\ & & \frac{\partial}{\partial x} \left( \mathcal{A} \frac{\partial \rho}{\partial x} \right) + \frac{\partial}{\partial y} \left( \mathcal{A} \frac{\partial \rho}{\partial y} \right) + \frac{\partial}{\partial z} \left( \kappa_E \frac{\partial \rho}{\partial z} \right) , \end{aligned}$$

### **Kinematic Boundary Conditions**

### flow cannot penetrate solid boundaries (e.g. land, or bottom topography) **impermeability condition**



**Figure 4-2** Notation and two physical interpretations of the bottom boundary condition illustrated here in a (x, z) plane for a topography independent of y. The impermeability of the bottom imposes that the velocity be tangent to the bottom defined by z - b = 0. In terms of the fluid budget, which can be extended to a finite volume approach, expressing that the horizontal inflow matches the vertical outflow requires u (b(x + dx) - b(x)) = w dx, which for  $dx \to 0$  leads to (4.28). Note that the velocity ratio w/u is equal to the topographic slope db/dx, which scales like the ratio of vertical to horizontal length scales, *i.e.*, the aspect ratio.

#### Volume flux balance at bottom

$$w = u \frac{\partial b}{\partial x} + v \frac{\partial b}{\partial y}$$

### **Kinematic Boundary Conditions**

flow cannot penetrate solid boundaries (e.g. land, or bottom topography) impermeability condition



**Reference** surface geoid

> Figure 4-3 Notation for the surface boundary condition. Expressing impermeability of the moving surface z –  $\eta = 0$  results in boundary condition (4.31). (The elevation of the sea surface height  $\eta$  is exaggerated compared to h for the purpose of illustration.)

Volume flux balance at surface

$$w = \frac{\partial \eta}{\partial t} + u \frac{\partial \eta}{\partial x} + v \frac{\partial \eta}{\partial y}$$
 at  $z = \eta$ .

### **Kinematic Boundary Conditions**

#### additional lateral boundary conditions for the free-surface



**Figure 4-4** Vertical section across an oceanic domain reaching the coast. Besides surface and bottom boundaries, the coast introduces an additional *lateral* boundary. Introducing an artificial vertical wall is necessary because a fixed numerical grid cannot describe well the exact position of the water's edge. Occasionally, a vertical wall is assumed at the shelf break, removing the entire shelf area from the domain, because the reduced physics of the model are incapable of representing some processes on the shelf.

ensuring a dynamic continuity at the boundaries

$$p_{\mathrm{atm}} = p_{\mathrm{sea}}$$
 at air-sea interface.  $z = \eta$ 

$$p_{\text{sea}}(z=0) = p_{\text{atm at sea level}} + \rho_0 g \eta \qquad z=0$$

ensuring a dynamic continuity at the boundaries

### Free Slip

(friction at the lateral boundary is ignored)



impermeability kinematic condition

### NO Slip

(friction at the lateral boundary is accounted for)



*dynamic condition velocity at boundary =0 --> Boundary Layer* 

ensuring a dynamic continuity at the boundaries



ensuring a dynamic continuity at the boundaries, for a moving boundary



ensuring a dynamic continuity at the boundaries, for a moving boundary



Continuity of velocity and tangential stresses

$$au^x = C_d \ 
ho_{air} \ U_{10} u_{10}, \quad au^y = C_d \ 
ho_{air} \ U_{10} v_{10},$$
 $U_{10} = \sqrt{u_{10}^2 + v_{10}^2}$ 
Cd= drag coefficient
Wind Speed at 10m

ensuring a dynamic continuity at the boundaries, for a open boundaries



**Figure 4-5** Open boundaries are common in regional modeling. Conditions at open boundaries are generally difficult to impose. In particular the nature of the condition depends on whether the flow enters the domain (carrying unknown information from the exterior) or leaves it (exporting known information). (*Courtesy of the HYCOM Consortium on Data-Assimilative Modeling*)

Nesting regional high resolution model raises the issues of prescribing boundary conditions in open ocean regions

#### **Tracer Boundary Conditions**

advective and diffusive fluxes of buoyancy (that is fluxes of heat and salt)

energy: 
$$\frac{\partial \rho}{\partial t} + u \frac{\partial \rho}{\partial x} + v \frac{\partial \rho}{\partial y} + w \frac{\partial \rho}{\partial z} = \frac{\partial}{\partial x} \left( \mathcal{A} \frac{\partial \rho}{\partial x} \right) + \frac{\partial}{\partial y} \left( \mathcal{A} \frac{\partial \rho}{\partial y} \right) + \frac{\partial}{\partial z} \left( \kappa_E \frac{\partial \rho}{\partial z} \right)$$

#### **Dirichlet condition**

prescribing the value of the variable (e.g. advective flux)

#### Newmann condition

prescribing the values of the gradient (e.g. diffusive flux)

#### **Cauchy condition, Robin condition**

prescribing the values of the total flux advective + diffusive **At a solid insulated boundary the total flux is set to zero.** 

#### **Tracer Boundary Conditions**

0

Ω

advective and diffusive fluxes of buoyancy (that is fluxes of heat and salt)

Ω

energy:

$$\frac{\partial \rho}{\partial t} + u \frac{\partial \rho}{\partial x} + v \frac{\partial \rho}{\partial y} + w \frac{\partial \rho}{\partial z} = \frac{\partial}{\partial x} \left( \mathcal{A} \frac{\partial \rho}{\partial x} \right) + \frac{\partial}{\partial y} \left( \mathcal{A} \frac{\partial \rho}{\partial y} \right) + \frac{\partial}{\partial z} \left( \kappa_E \frac{\partial \rho}{\partial z} \right)$$

#### Example of surface Boundary Condition

$$-\kappa_T \left. \frac{\partial T}{\partial z} \right|_{z=\eta} = F(T_{\text{sea}}, T_{\text{air}}, \mathbf{u}_{10}, \text{cloudiness, moisture}, ...)$$

#### **Dirichlet condition**

prescribing the value of the variable (e.g. advective flux)

 $\Delta x$ 

$$\tilde{T}_m = T_b$$



#### Newmann condition

prescribing the values of the gradient (e.g. diffusive flux)



#### **Cauchy condition, Robin condition**

prescribing the values of the total flux advective + diffusive



higher order operators ..



Figure 4-7 An operator spanning 2 points on each side of the calculation point can be applied only up to m - 2 if a single Dirichlet condition is prescribed. When applying the same operator at m - 1 we face the problem that the value at m + 1 does not exist.

### **Accuracy and Errors**

- *Modeling errors*: This error is caused by the imperfections of the mathematical model in representing the physical system. It is thus the difference between the evolution of the real system and that of the exact solution of its mathematical representation. Earlier in this chapter we introduced simplifications to the equations and added parameterizations of unresolved processes, which all introduce errors of representation. Furthermore, even if the model formulation had been ideal, coefficients remain imperfectly known. Uncertainties in the accompanying boundary conditions also contribute to modeling errors.
- *Discretization errors*: This error is introduced when the original equations are approximated to transform them into a computer code. It is thus the difference between the exact solution of the continuous problem and the exact numerical solution of the discretized equations. Examples are the replacement of derivatives by finite differences and the use of guesses in predictor-corrector schemes.
- *Iteration errors*: This error originates with the use of iterative methods to perform intermediate steps in the algorithm and is thus measured as the difference between the exact solution of the discrete equations and the numerical solution actually obtained. An example is the use of the so-called Jacobi method to invert a matrix at some stage of the calculations: for the sake of time, the iterative process is interrupted before full convergence is reached.
- *Rounding errors*: These errors are due to the fact that only a finite number of digits are used in the computer to represent real numbers.

A well constructed model should ensure that

rounding errors  $\ll$  iteration errors  $\ll$  discretization errors  $\ll$  modeling errors.



Figure 8-3 Frictional influence of a flat bottom on a uniform flow in a rotating framework.

x -momentum:

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} - fv = -\frac{1}{\rho_0} \frac{\partial p}{\partial x} + \frac{\partial}{\partial x} \left( \mathcal{A} \frac{\partial u}{\partial x} \right) + \frac{\partial}{\partial y} \left( \mathcal{A} \frac{\partial u}{\partial y} \right) + \frac{\partial}{\partial z} \left( \nu_E \frac{\partial u}{\partial z} \right)$$

surface and bottom boundary conditions for vertical derivative

x -momentum:

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} - fv = -\frac{1}{\rho_0} \frac{\partial p}{\partial x} + \frac{\partial}{\partial x} \left( \mathcal{A} \frac{\partial u}{\partial x} \right) + \frac{\partial}{\partial y} \left( \mathcal{A} \frac{\partial u}{\partial y} \right) + \frac{\partial}{\partial z} \left( \nu_E \frac{\partial u}{\partial z} \right)$$

surface and bottom boundary conditions for vertical derivative

$$\begin{array}{ccc} inertial \ terms & \textit{rotation} & \textit{frictional forces} \\ \hline 1 \\ \Omega T \end{array}, \quad \frac{U}{\Omega L} \ , \quad \frac{U}{\Omega L} \ , \quad \frac{WL}{UH} \ \cdot \ \frac{U}{\Omega L} \end{array} \right|, \quad 1 \ , \quad \frac{P}{\rho_0 \Omega L U} \ , \quad \frac{\mathcal{A}}{\Omega L^2} \ , \quad \frac{\nu_E}{\Omega H^2} \end{array}$$

Ekman Number

viscous force/rotation



inertial terms rotation

Ekman number typically small and friction is neglected, except close to the boundary ~1

 $rac{U}{\Omega L}\,,\quad rac{U}{\Omega L}\,,\quad rac{WL}{UH}\,\cdot\, rac{U}{\Omega L}\,,$ 

*Ekman Number* viscous force/rotation





#### Dominant balance is between *rotation* and *friction terms*

rotationfrictional forces,1,
$$\frac{P}{\rho_0 \Omega L U}$$
, $\frac{\mathcal{A}}{\Omega L^2}$ , $\frac{\nu_E}{\Omega H^2}$ 



x - momentum:

Dominant balance is between *rotation* and *vertical friction terms* 

rotationfrictional forces1
$$\frac{P}{\rho_0 \Omega L U}$$
 $\frac{\mathcal{A}}{\Omega L^2}$  $\frac{\nu_E}{\Omega H^2}$ 

$$-f(v-\bar{v}) = \nu_E \frac{\partial^2 u}{\partial z^2} +f(u-\bar{u}) = \nu_E \frac{\partial^2 v}{\partial z^2} Surface (z = 0): \qquad \rho_0 \nu_E \frac{\partial u}{\partial z} = \tau^x, \ \rho_0 \nu_E \frac{\partial v}{\partial z} = \tau^y Toward interior (z \to -\infty): \qquad u = \bar{u}, \ v = \bar{v}.$$



Figure 8-6 The surface Ekman layer generated by a wind stress on the ocean.

$$-f(v-\bar{v}) = \nu_E \frac{\partial^2 u}{\partial z^2} +f(u-\bar{u}) = \nu_E \frac{\partial^2 v}{\partial z^2} Surface (z = 0): \qquad \rho_0 \nu_E \frac{\partial u}{\partial z} = \tau^x, \ \rho_0 \nu_E \frac{\partial v}{\partial z} = \tau^y Toward interior (z \to -\infty): \qquad u = \bar{u}, \ v = \bar{v}.$$

$$-f(v-\bar{v}) = \nu_E \frac{\partial^2 u}{\partial z^2} +f(u-\bar{u}) = \nu_E \frac{\partial^2 v}{\partial z^2} Surface (z = 0): \qquad \rho_0 \nu_E \frac{\partial u}{\partial z} = \tau^x, \ \rho_0 \nu_E \frac{\partial v}{\partial z} = \tau^y Toward interior (z \to -\infty): \qquad u = \bar{u}, \ v = \bar{v}.$$

#### Ekman Transport

$$U = \int_{-\infty}^{0} (u - \bar{u}) dz = \frac{1}{\rho_0 f} \tau^y$$
$$V = \int_{-\infty}^{0} (v - \bar{v}) dz = \frac{-1}{\rho_0 f} \tau^x$$

$$-f(v-\bar{v}) = \nu_E \frac{\partial^2 u}{\partial z^2} +f(u-\bar{u}) = \nu_E \frac{\partial^2 v}{\partial z^2} Surface (z = 0): \qquad \rho_0 \nu_E \frac{\partial u}{\partial z} = \tau^x, \ \rho_0 \nu_E \frac{\partial v}{\partial z} = \tau^y Toward interior (z \to -\infty): \qquad u = \bar{u}, \ v = \bar{v}.$$

$$\frac{\text{Solution}}{u} = \bar{u} + \frac{\sqrt{2}}{\rho_0 f d} e^{z/d} \left[ \tau^x \cos\left(\frac{z}{d} - \frac{\pi}{4}\right) - \tau^y \sin\left(\frac{z}{d} - \frac{\pi}{4}\right) \right]$$
$$v = \bar{v} + \frac{\sqrt{2}}{\rho_0 f d} e^{z/d} \left[ \tau^x \sin\left(\frac{z}{d} - \frac{\pi}{4}\right) + \tau^y \cos\left(\frac{z}{d} - \frac{\pi}{4}\right) \right]$$



$$v = \bar{v} + \frac{\sqrt{2}}{\rho_0 f d} e^{z/d} \left[ \tau^x \sin\left(\frac{z}{d} - \frac{\pi}{4}\right) + \tau^y \cos\left(\frac{z}{d} - \frac{\pi}{4}\right) \right]$$

#### Implication of the Ekman surface Layers



Figure 8-8 Ekman pumping in an ocean subject to sheared winds (case of Northern Hemisphere).



Figure 8-3 Frictional influence of a flat bottom on a uniform flow in a rotating framework.

$$-fv = -\frac{1}{\rho_0} \frac{\partial p}{\partial x} + \nu_E \frac{\partial^2 u}{\partial z^2}$$

$$+fu = -\frac{1}{\rho_0} \frac{\partial p}{\partial y} + \nu_E \frac{\partial^2 v}{\partial z^2}$$

$$0 = -\frac{1}{\rho_0} \frac{\partial p}{\partial z} ,$$

Bottom (z = 0):  $u = 0, \quad v = 0,$ Toward the interior  $(z \gg d)$ :  $u = \bar{u}, \quad v = 0, \quad p = \bar{p}(x, y).$ 



**Figure 8-4** The velocity spiral in the bottom Ekman layer. The figure is drawn for the Northern Hemisphere (f > 0), and the deflection is to the left of the current above the layer. The reverse holds for the Southern Hemisphere.